

# Closed-Loop Control of a Combustion Engine Model

Loay Altaweel

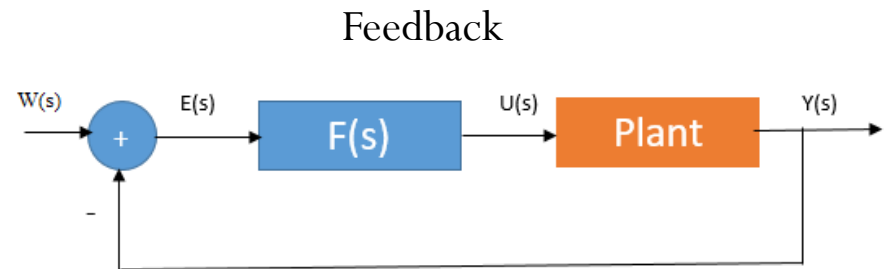
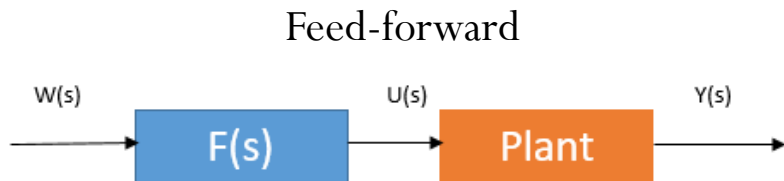
Supervised by Dr. Waschl Harald

# Agenda

- Background
- Controllers
- Problem
- Goals
- Controller choice
- Controller analysis and design
- Conclusion

# Background

Feed-forward & Feedback control loops (Plant  $\rightarrow P(s)$ )



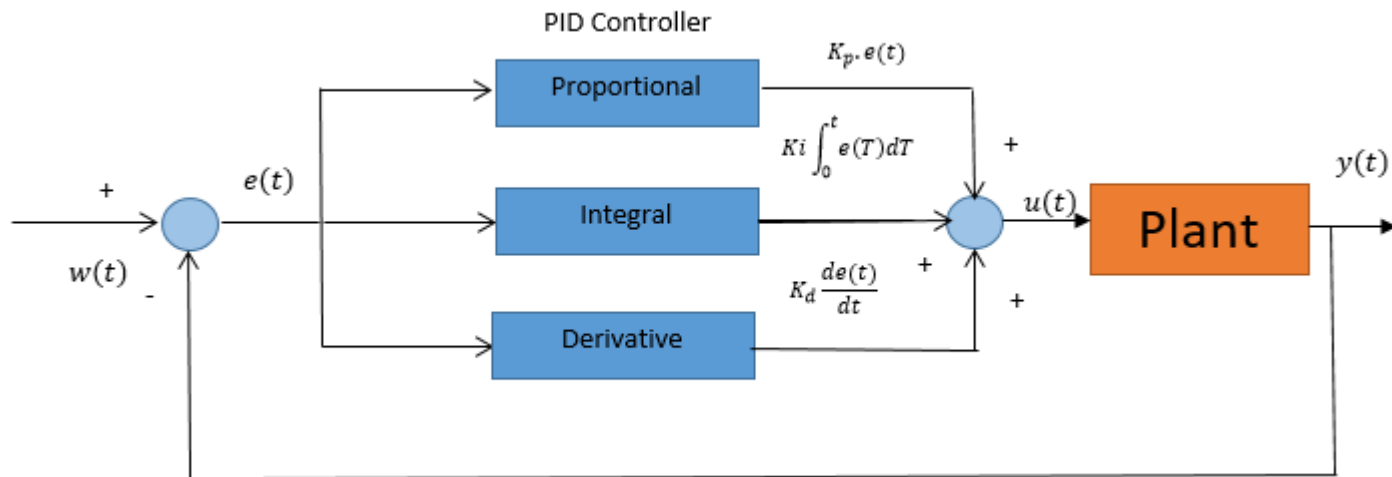
$$F(s) = P(s)^{-1}$$

$$F(s)P(s) \text{ very high}$$

$$G(s) = \frac{F(s)P(s)}{1 + F(s)P(s)}$$

# Controllers: Proportional-Integral-Derivative (PID)

$$u(t) = K_p \cdot e(t) + K_i \cdot \int_0^t e(T) dT + K_d \frac{d.e(t)}{dt}$$



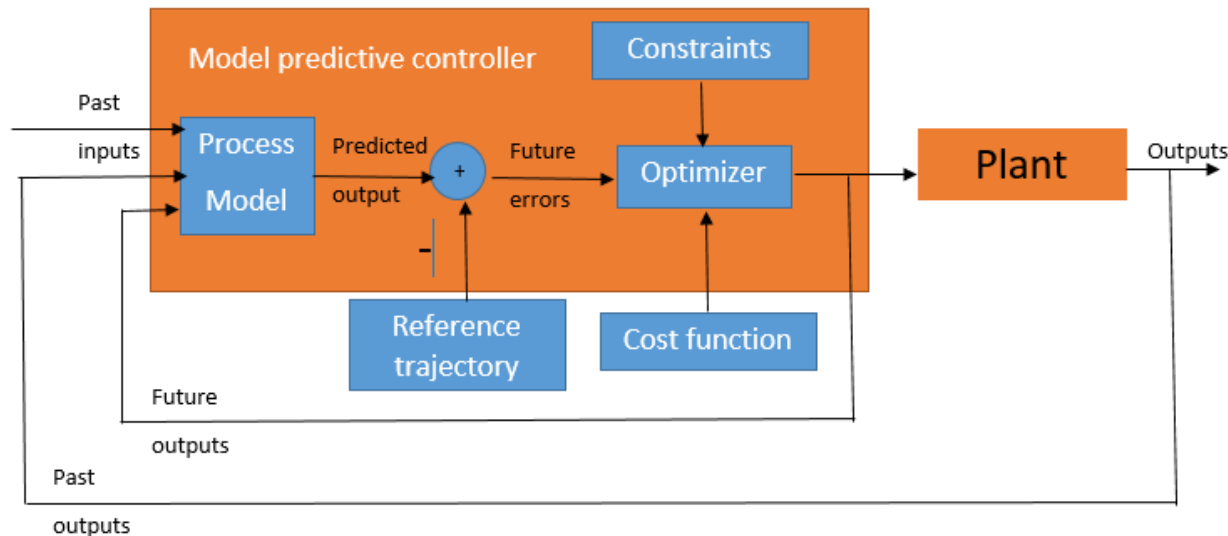
# Controllers: Model Predictive control (MPC)

$$\min_{\mathbf{u}} \sum_{j=k}^{k+N-1} (x^T(j|k)Qx(j|k) + u^T(j|k)Ru(j|k))$$

subject to

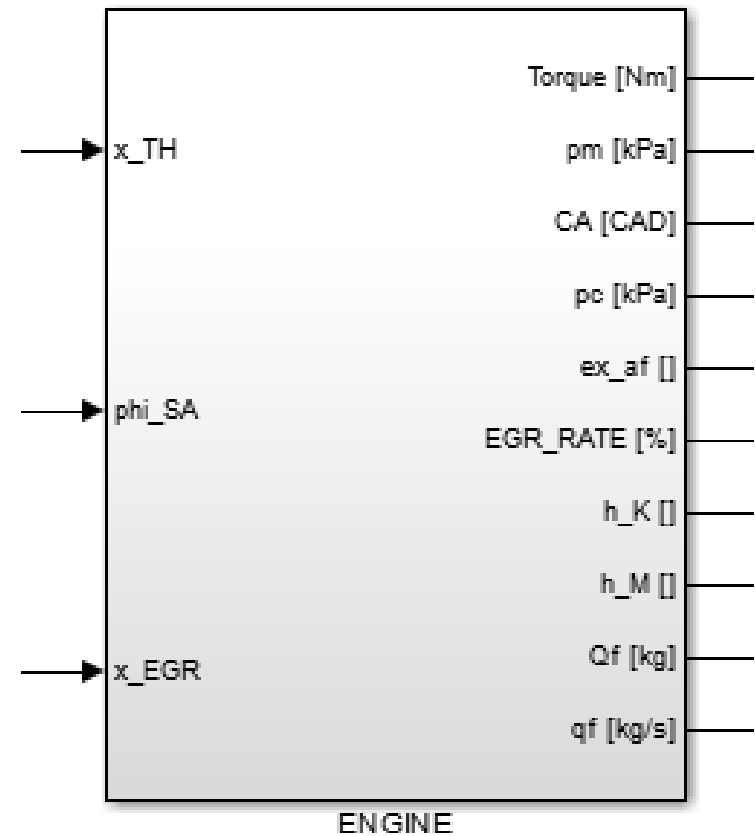
$$u(k+j|k) \in U$$

$$x(k+j|k) = Ax(k+j-1|k) + Bu(k+j-1|k)$$



# Problem

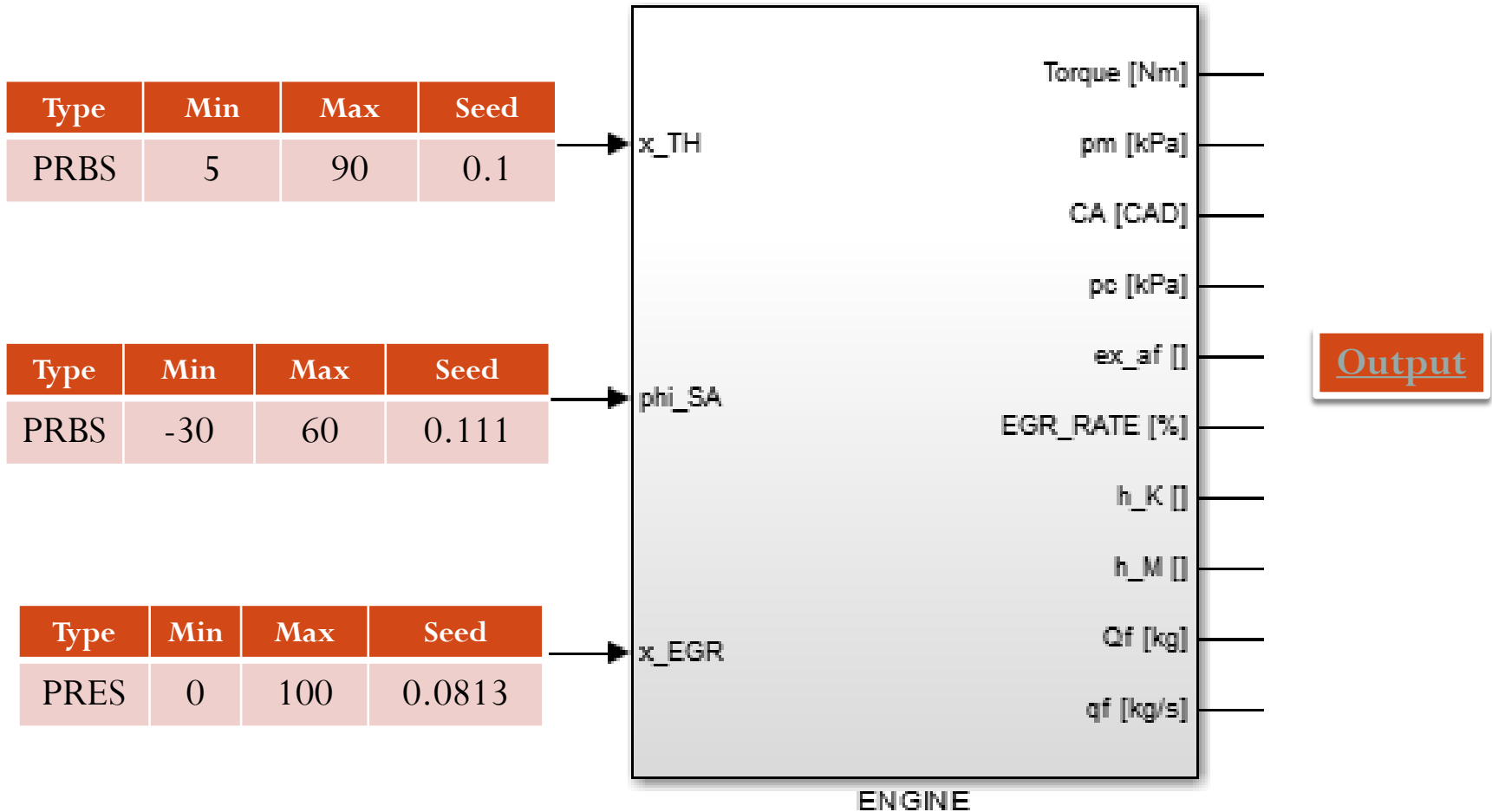
Design an optimal closed-loop control for a **gasoline** combustion engine taking into consideration actuators and limitations of the system (knocking & misfiring).



# Goals

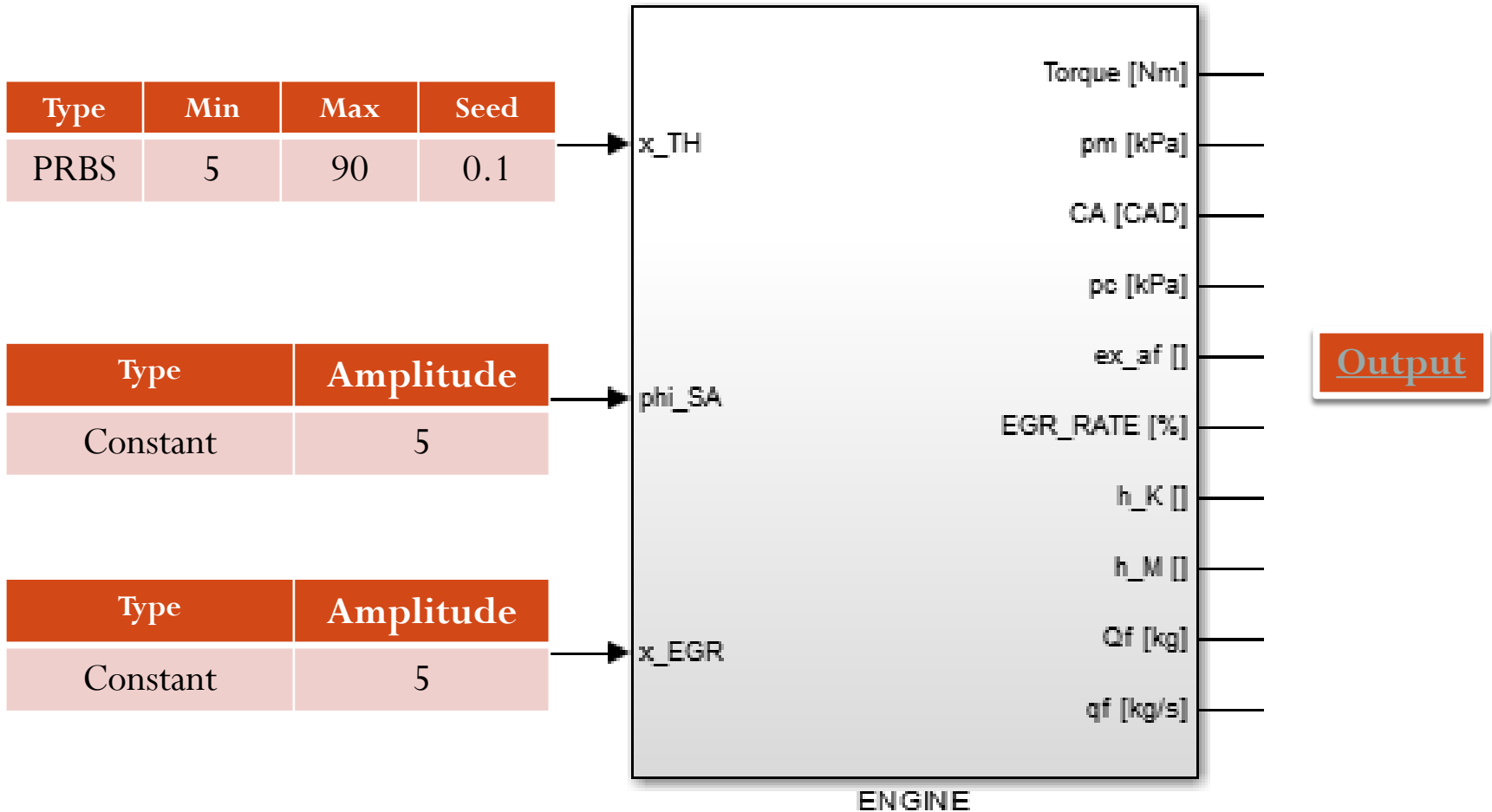
- The main goal is to increase the efficiency of the engine model while following the desired torque trajectory
- Achieve the following desirable outputs:
  1. Reducing knocking and misfiring.
  2. Adjusting the EGR valve position to maximize energy output of the engine.
  3. Minimizing the over all fuel consumption.

# Simulation Scenarios: ONE





# Simulation Scenarios: TWO

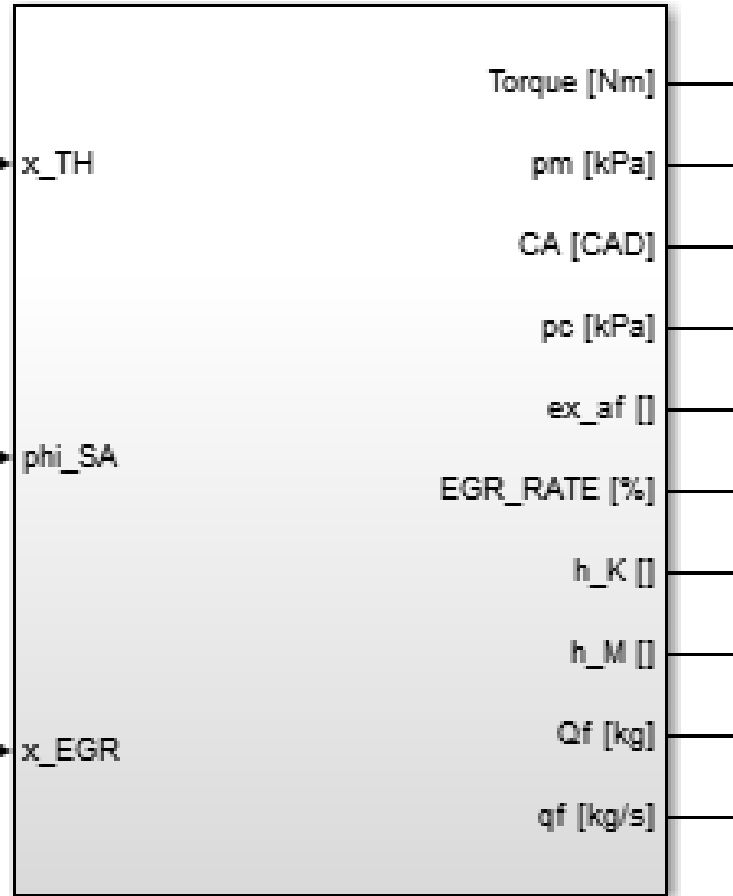


# Simulation Scenarios: **THREE**

Type	Min	Max	Step time
Step	5	90	30

Type	Min	Max	Seed
PRBS	-30	60	0.111

Type	Amplitude
Constant	5



**Output**

ENGINE

# Simulation Scenarios: **FOUR**

Type	Min	Max	Step time
Step	5	90	30

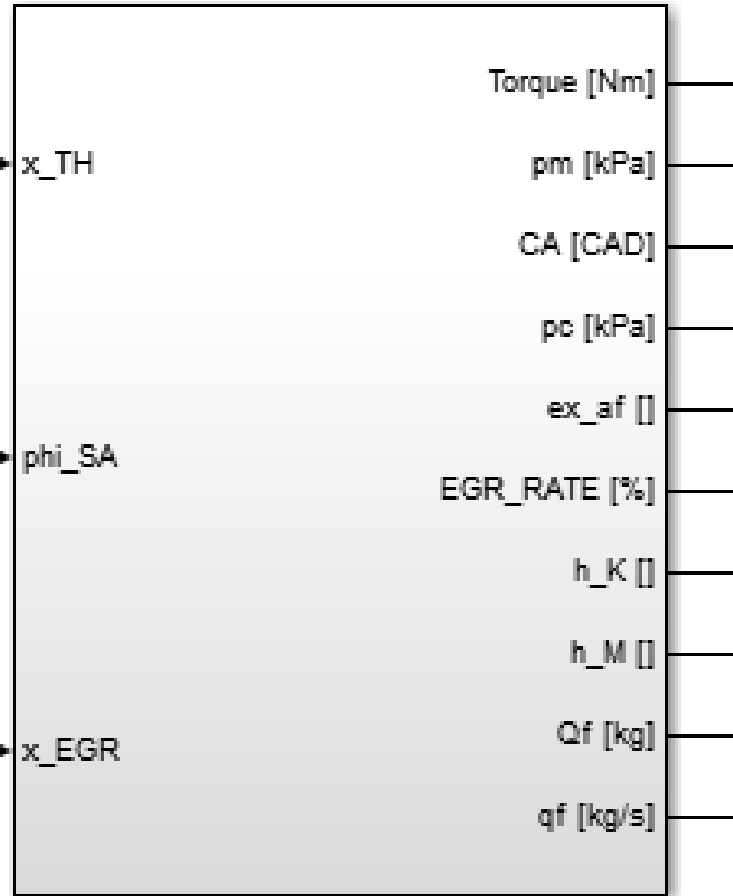
x\_TH

Type	Amplitude
Constant	60

phi\_SA

Type	Min	Max	Seed
PRBS	-30	60	0.111

x\_EGR



**Output**

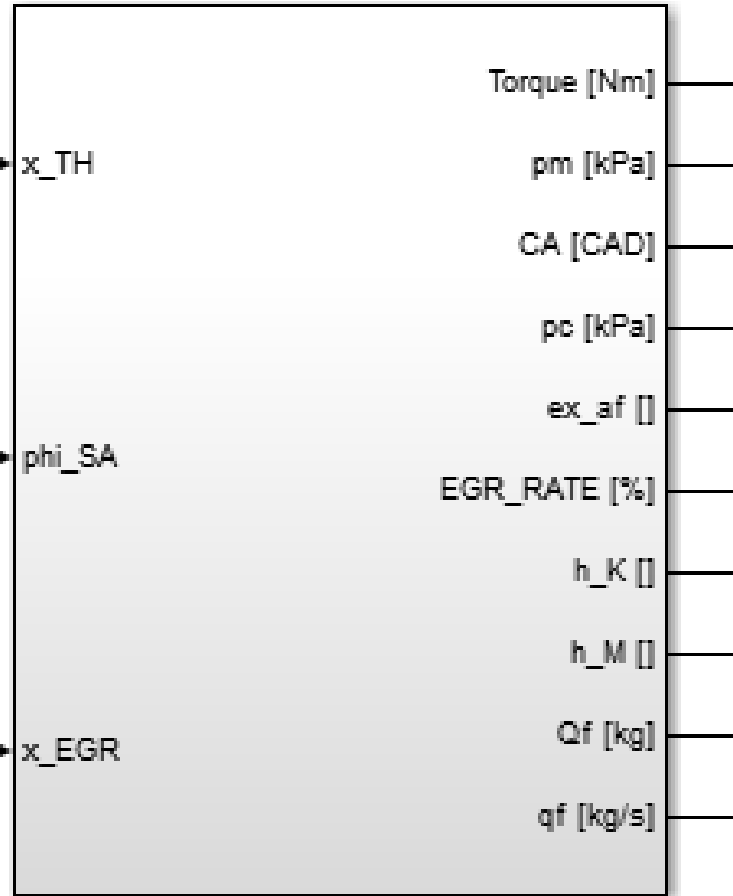
ENGINE

# Simulation Scenarios: FIVE

Type	Step width	Initial value	Step height
Step	10	5	2

Type	Amplitude
Constant	60

Type	Amplitude
Constant	100



**Output**

ENGINE

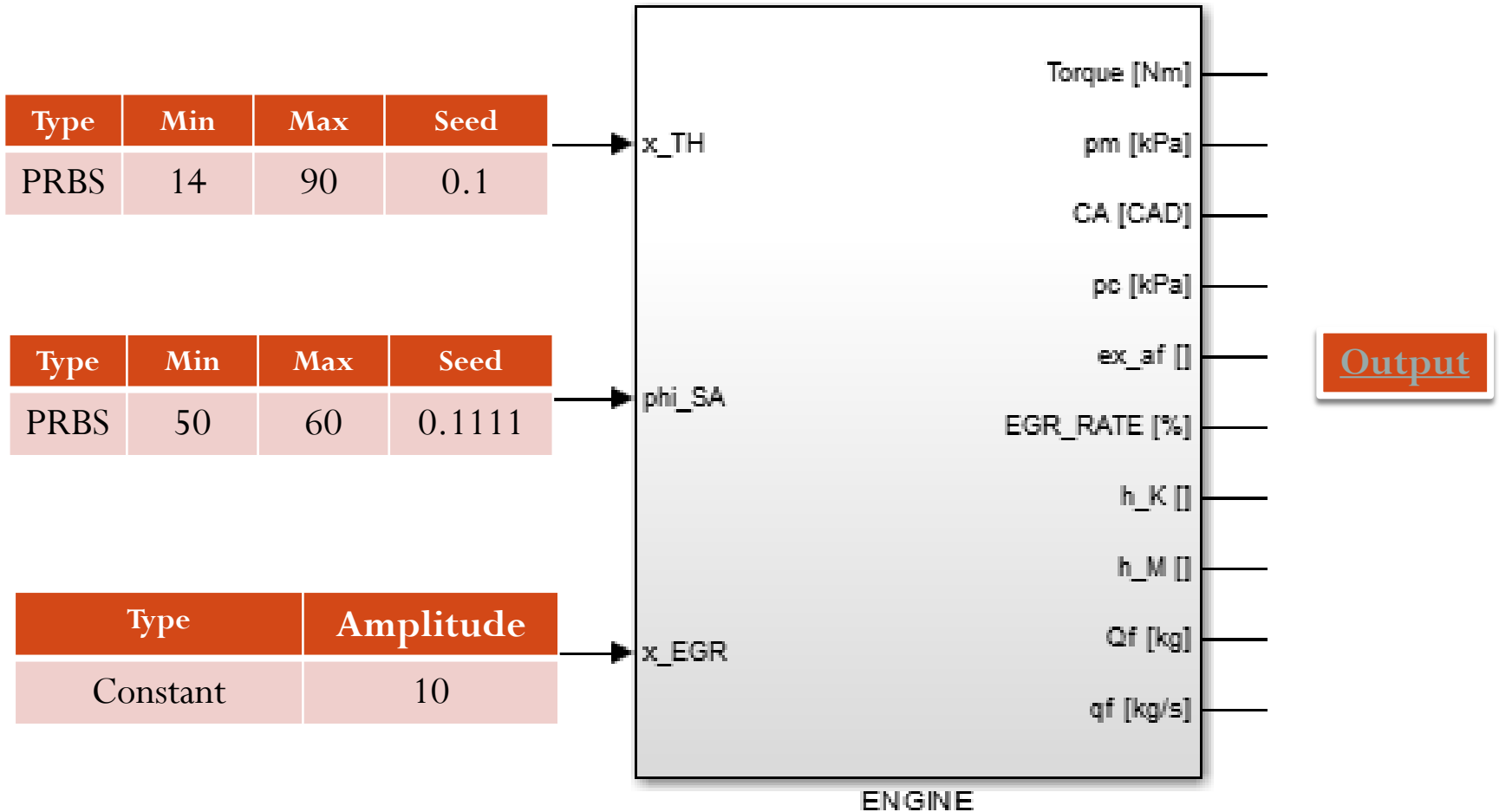
# Conclusion of Scenarios **ONE-FIVE**

- Scenarios revealed strong coupling between inputs and outputs
  - Henceforth, MPC controller should be used.

# MPC Controller Design: Steps

1. Linearization of the provided model (Toyota model)
2. Defining horizons options.
3. Define input and output constraints.
4. Adjusting output and input weights, and input rate weight to accommodate the non linear plant to the desired output.

# Identification



# Identification: Result

- Cylinders 1 and 2 have been selected as basis for an MPC controller design.
  - They have the highest fit values.

output	cylinder 1	cylinder 2	cylinder 3	cylinder 4
torque	76.88	79.51	-	72.96
knocking	46.73	64.29	-	51.95
misfiring	24.33	37.19	-	21.41
fuel consumption	76.38	78.28	-	69.79
system order	2	5	-	2



# MPC Controller Design: Controller

## MPC<sub>c1,basic</sub>

- MPC<sub>c1,basic</sub>: designed based on **Cylinder 1**.

Name	Weight	Rate Weight
Throttle	0	8.8674
Spark Advance	0	5.5116
EGR position	0	13.2278

mi

Table 8: Inputs Weights for nonlinear model

Name	Weight
Torque	0.22679
Knocking	0.49895
Misfiring	0.63503
Fuel consumption	0.0028804

mo

Table 9: Outputs Weights for nonlinear model

# MPC Controller Design: Controller

## MPC<sub>c1,advanced</sub>

- MPC<sub>c1,advanced</sub>: based on **Cylinder 1** with error adjustment.

Name	Weight	Rate Weight
Throttle	0	8.8674
Spark Advance	0	5.5116
EGR position	0	13.2278

mi

Table 8: Inputs Weights for nonlinear model

Name	Weight
Torque	0.22679
Knocking	0.49895
Misfiring	0.63503
Fuel consumption	0.0028804

mo

Table 9: Outputs Weights for nonlinear model

signal name	output disturbances type	output disturbances magnitude	Measurement Noise type	Measurement Noise magnitude
Torque	step	10	white	1
Knocking	step	0.01	white	1
Misfire	step	0.01	white	1
Fuel consumption	step	$1e^{-5}$	white	1

Table 10: Type and magnitude of the error signals.

# MPC Controller Design: Controller

## MPC<sub>c2,advanced</sub>

- MPC<sub>c2,advanced</sub> : based on **Cylinder 2** with error adjustment.

Name	Weight	Rate Weight
Throttle	0	48.3416
Spark Advance	0.15386	148.6233
EGR position	0.0017311	3.4684

mi

Table 11: Inputs Weights for nonlinear model

Name	Weight
Torque	0.74961
Knocking	0.029526
Misfiring	0.12397
Fuel consumption	0.025704

mo

Table 12: Outputs Weights for nonlinear model

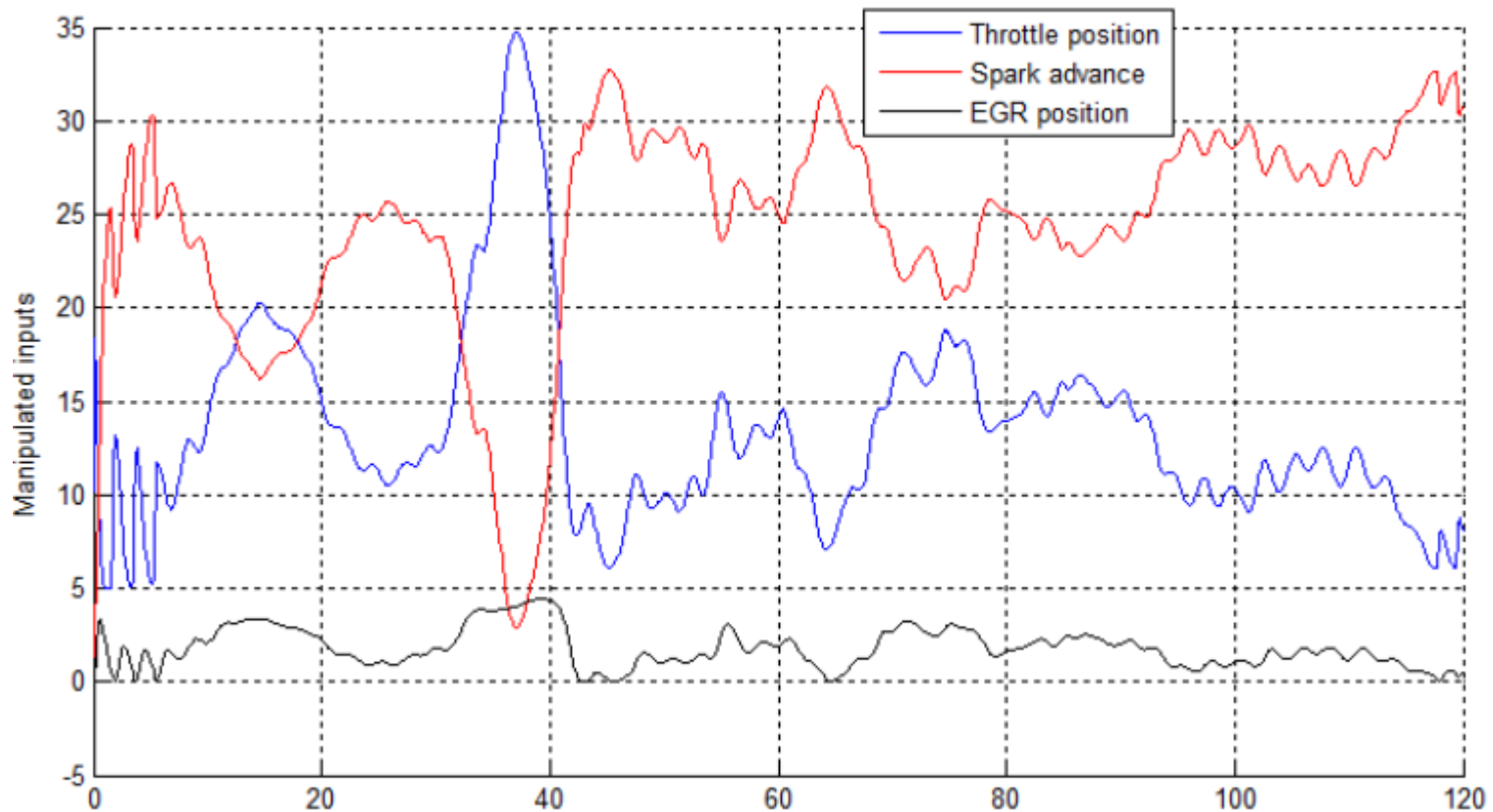
signal name	output disturbances type	output disturbances magnitude	Measurement Noise type	Measurement Noise magnitude
Torque	step	10	white	1
Knocking	step	0.01	white	1
Misfire	step	0.01	white	1
Fuel consumption	step	$1e^{-5}$	white	1

Table 10: Type and magnitude of the error signals.

# MPC Controller Design: Evaluation of

## MPC<sub>c1,basic</sub>

- Input

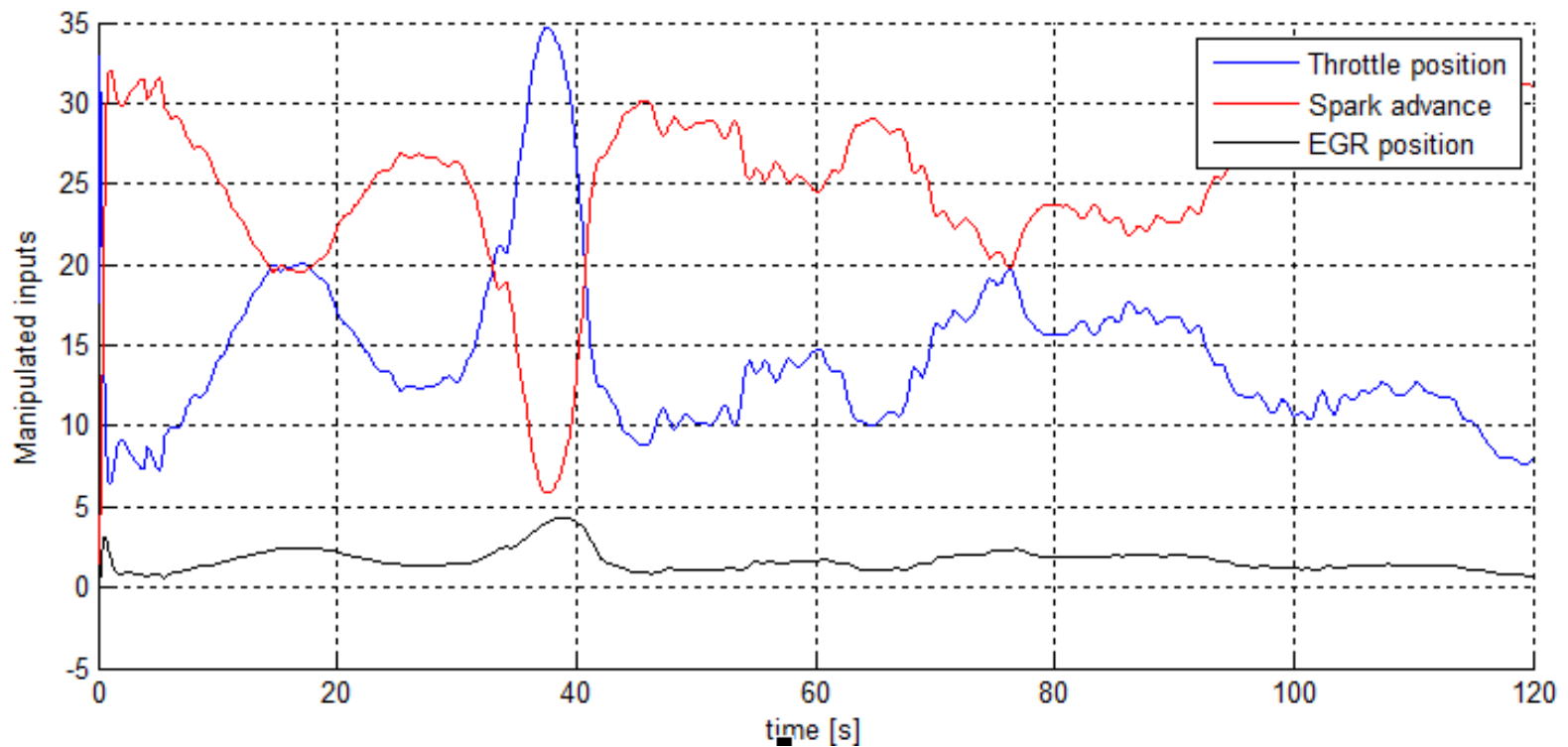


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# MPC Controller Design: Evaluation of

## MPC<sub>c1,advanced</sub>

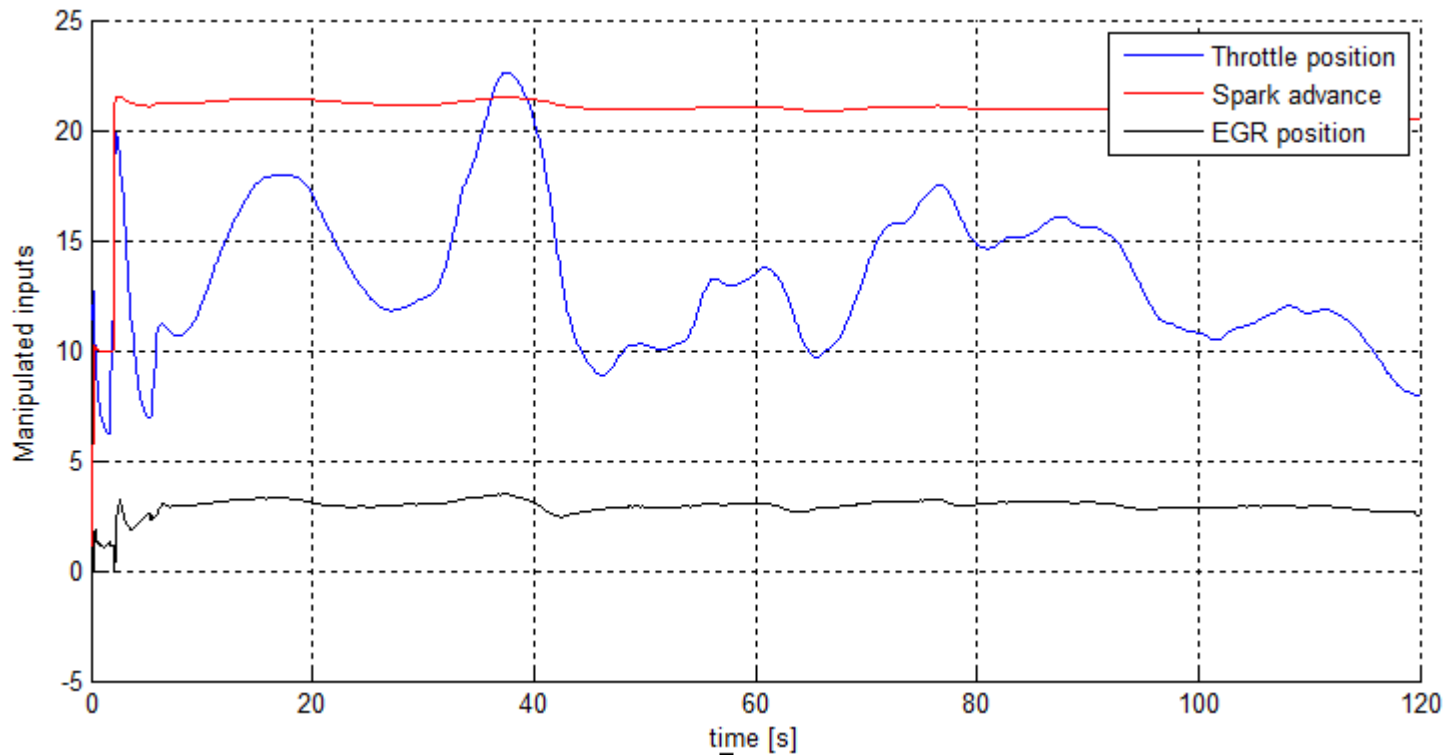
- Input



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# MPC Controller Design: Evaluation of $MPC_{c2,advanced}$

- Input

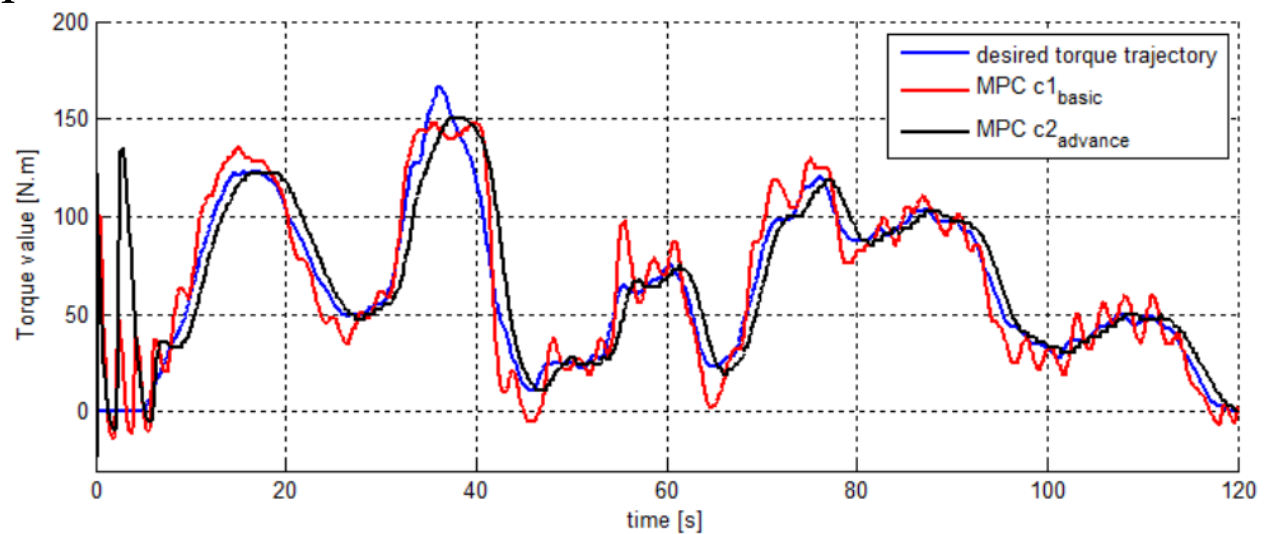


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# Conclusion

- We have selected **MPC<sub>c2,advanced</sub>** because it has the highest efficiency while having a smooth torque trajectory.

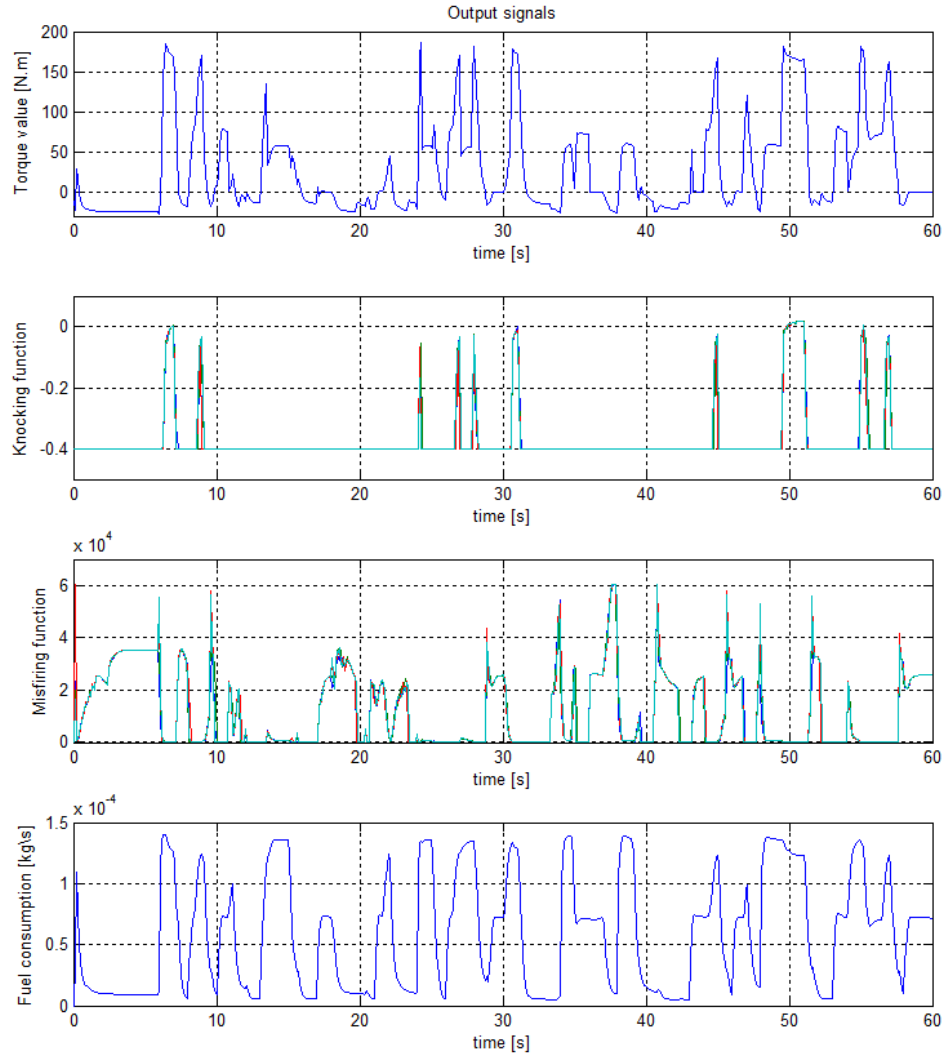
Signals	$MPC_{c1\_basic}$	$MPC_{c1\_advanced}$	$MPC_{c2\_advanced}$
torque	126.0657	28.025	138.1639
knocking	0	0	0
misfire	0.2681	0	9.7705
fuel consumption	$4.2256 * 10^{-9}$	$8.1545 * 10^{-9}$	$4.1255 * 10^{-9}$
throttle	205.8399	226.5927	182.9434
spark advance	606.4382	606.2328	424.0316
EGR value	4.2121	3.1248	8.5438
$\bar{\mu}_{engine}$	28.84	12.20	30.77



**Questions !**

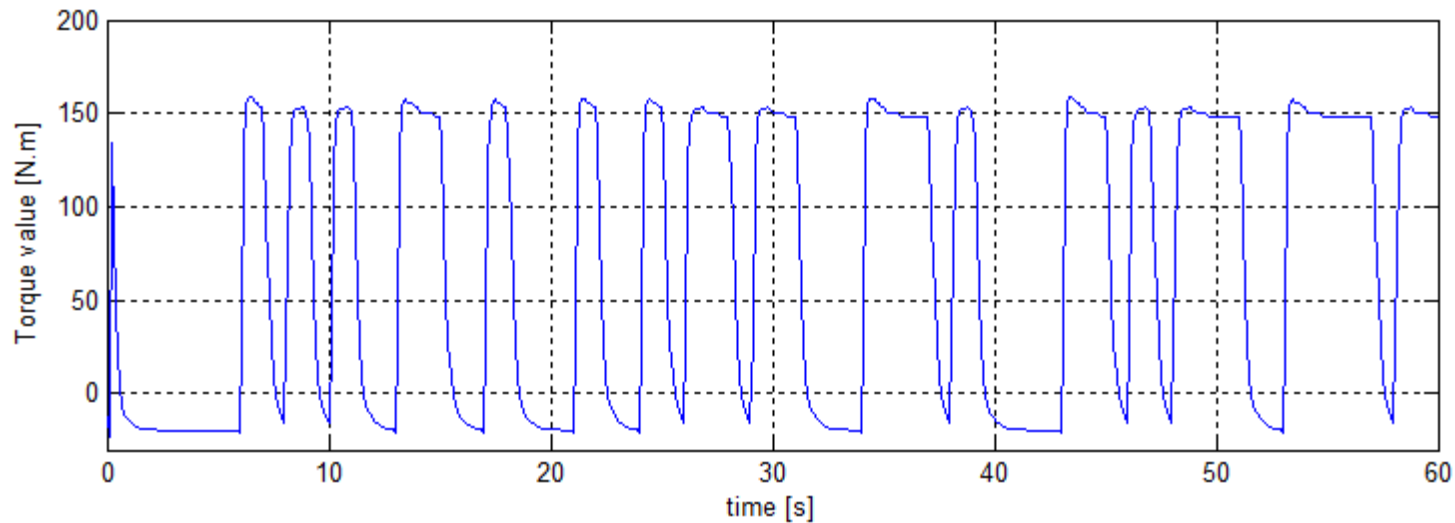
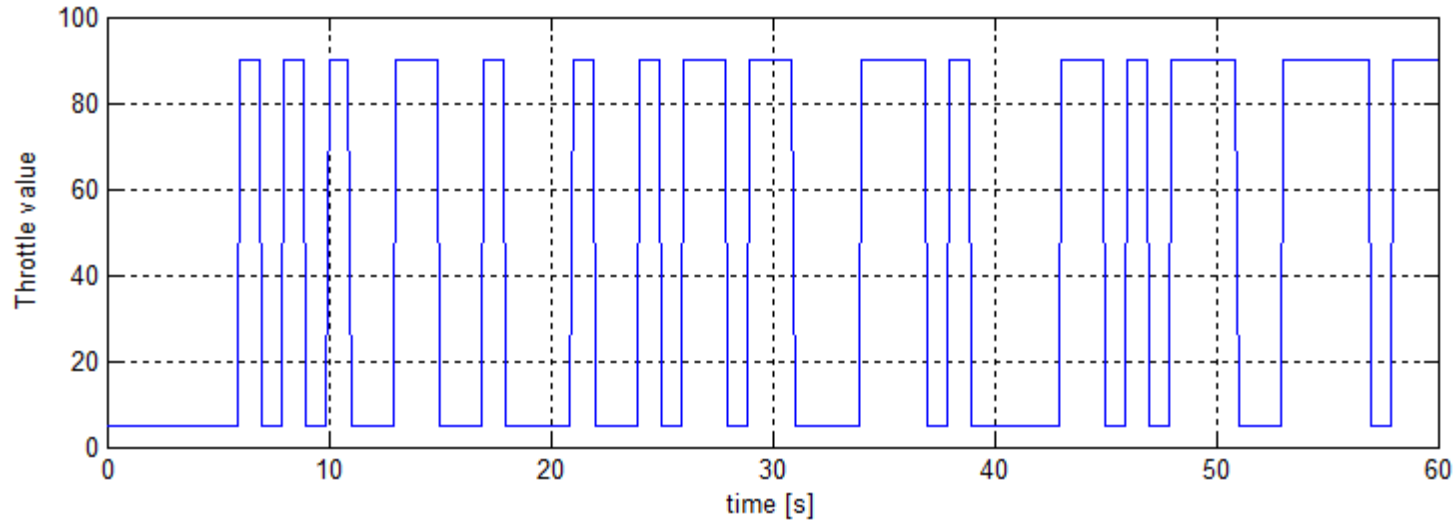


# Scenario ONE: output



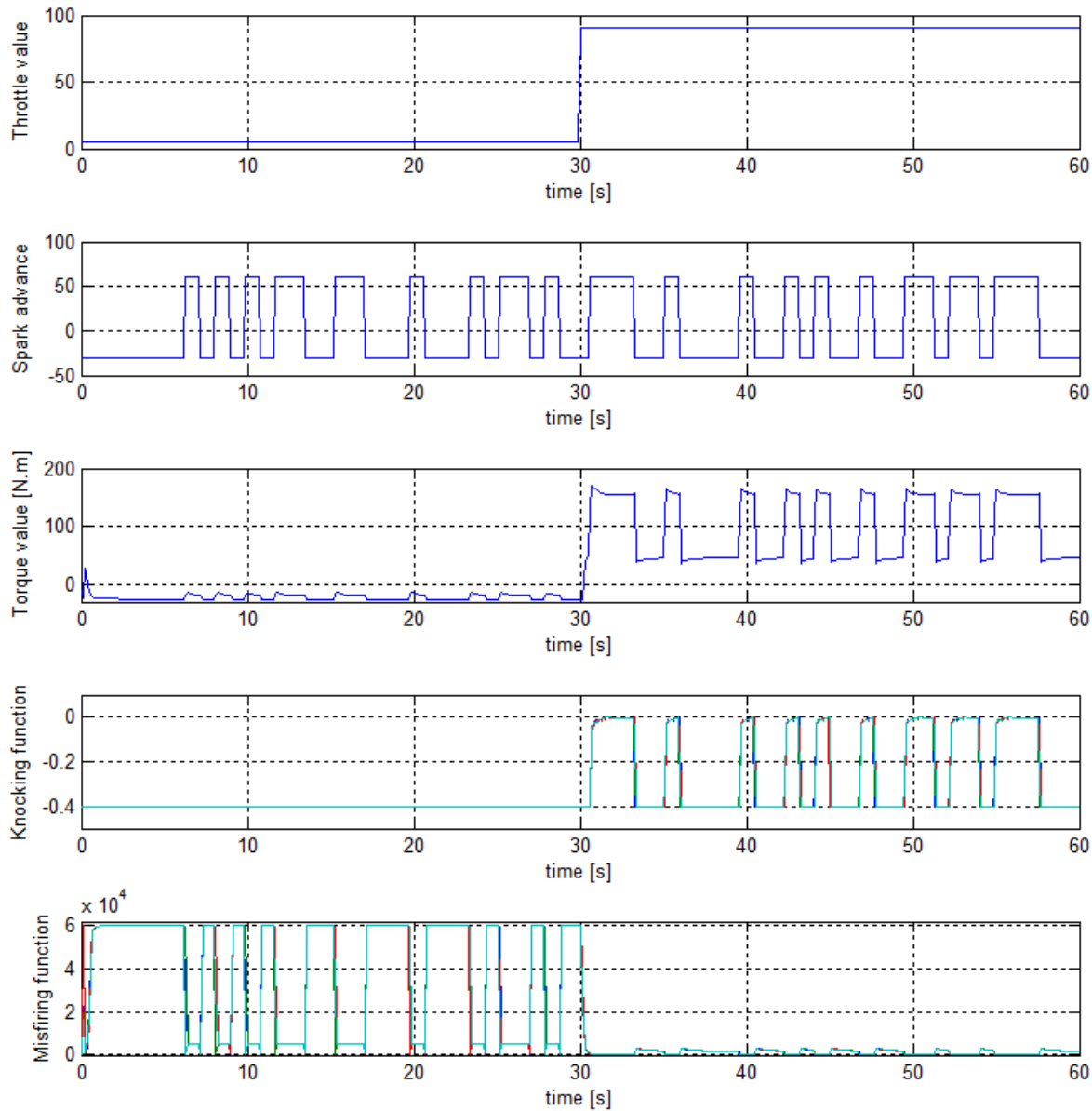
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# Scenario TWO: output



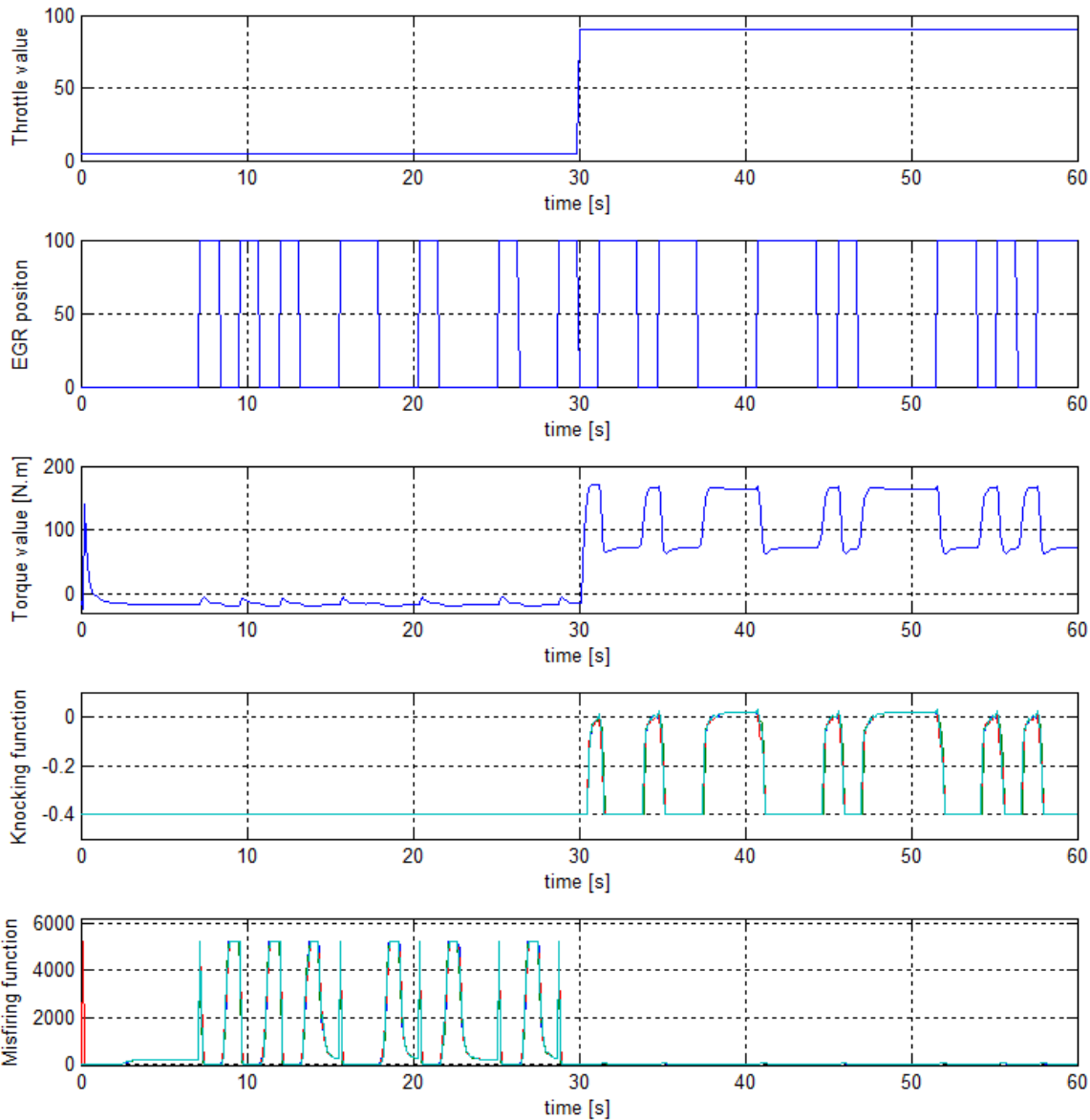
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# Scenario **THREE**: output



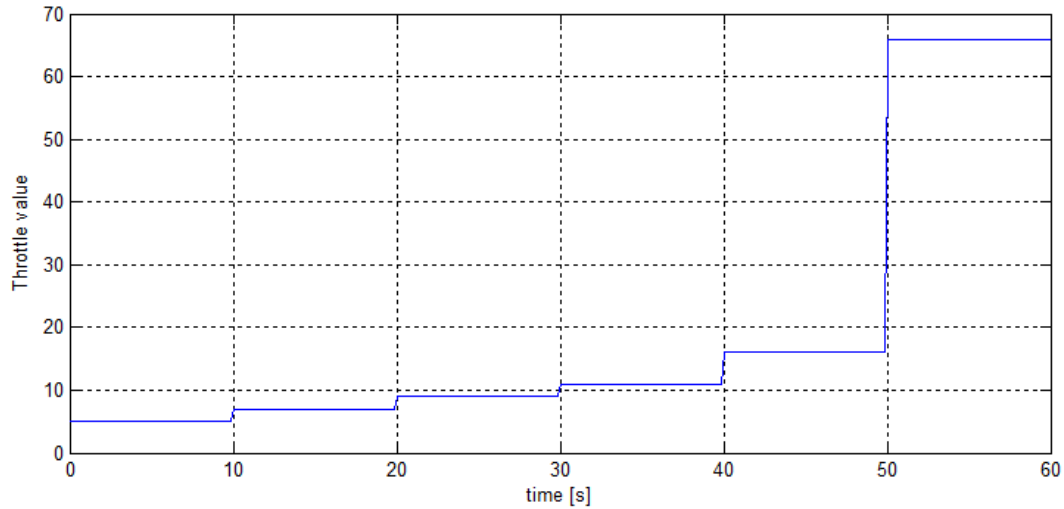
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# Scenario **FOUR**: output

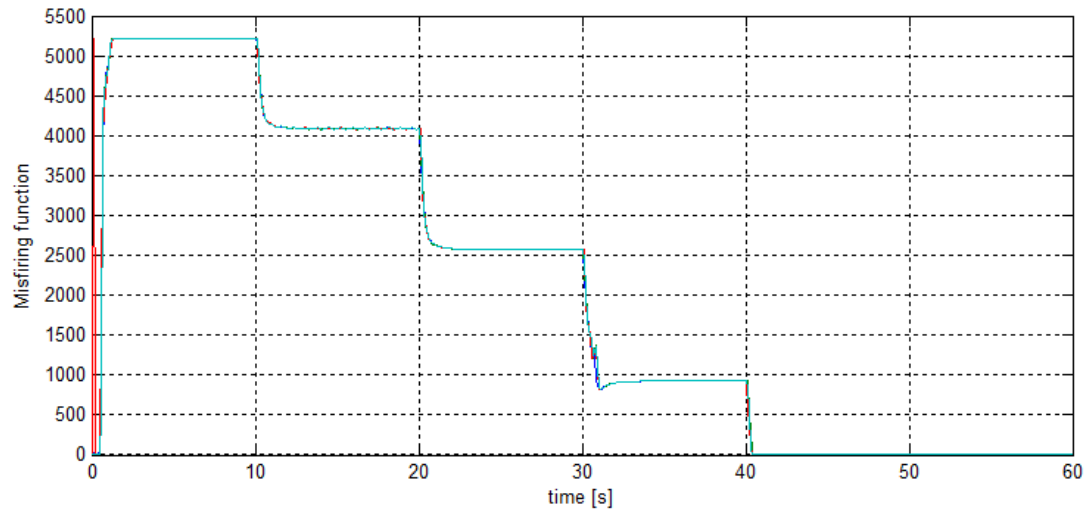


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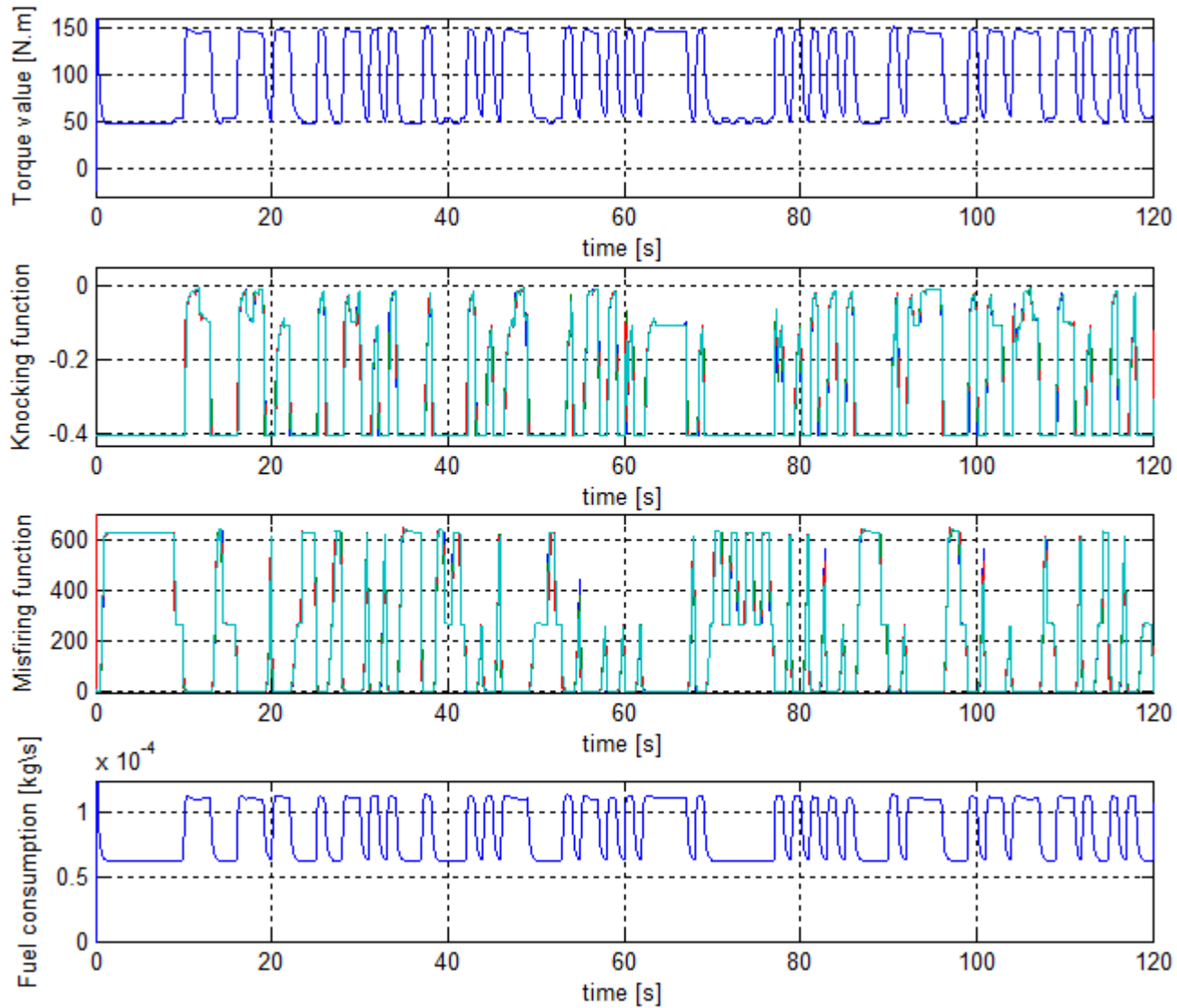
# Scenario FIVE: output



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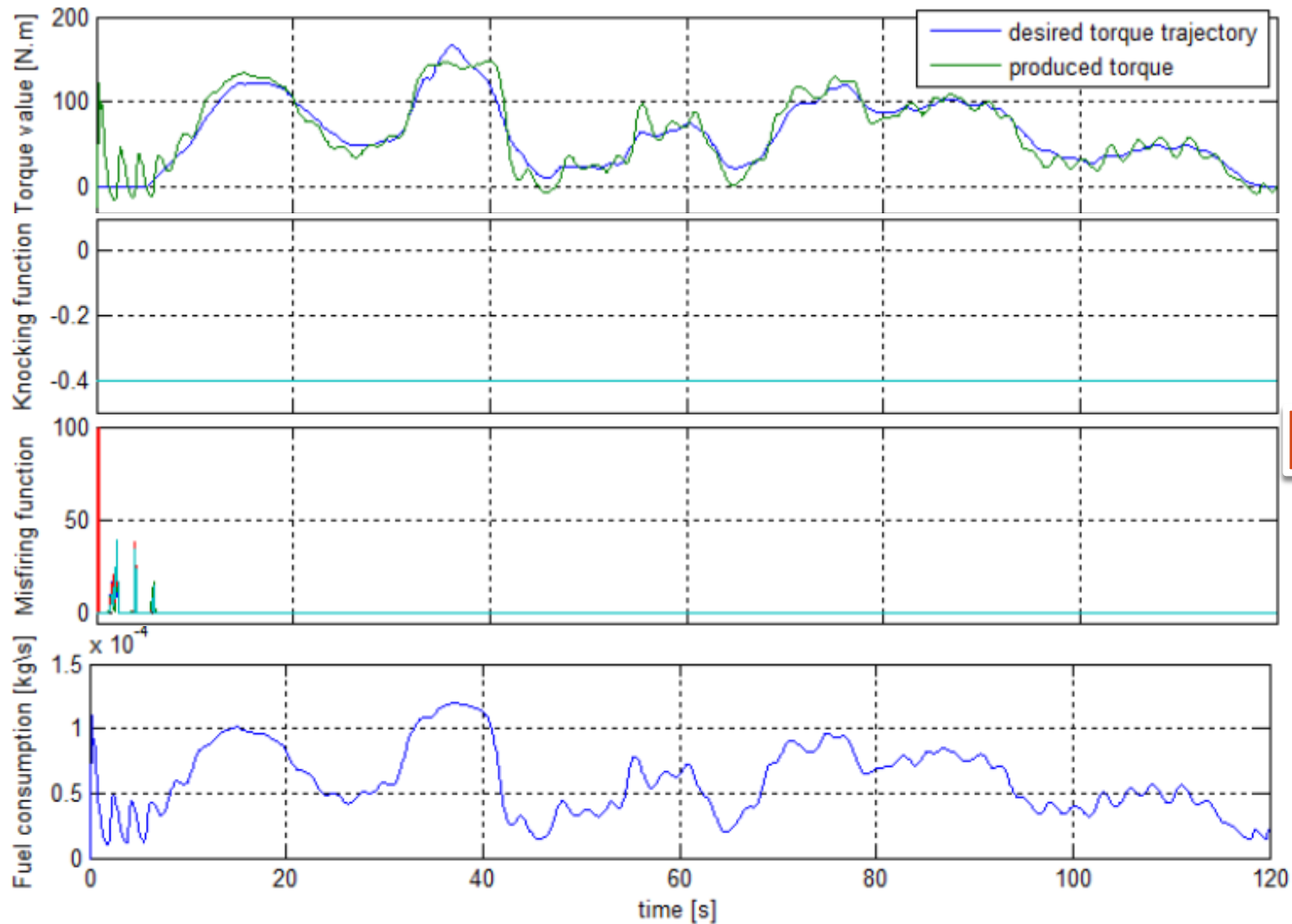


# Identification Scenario: output



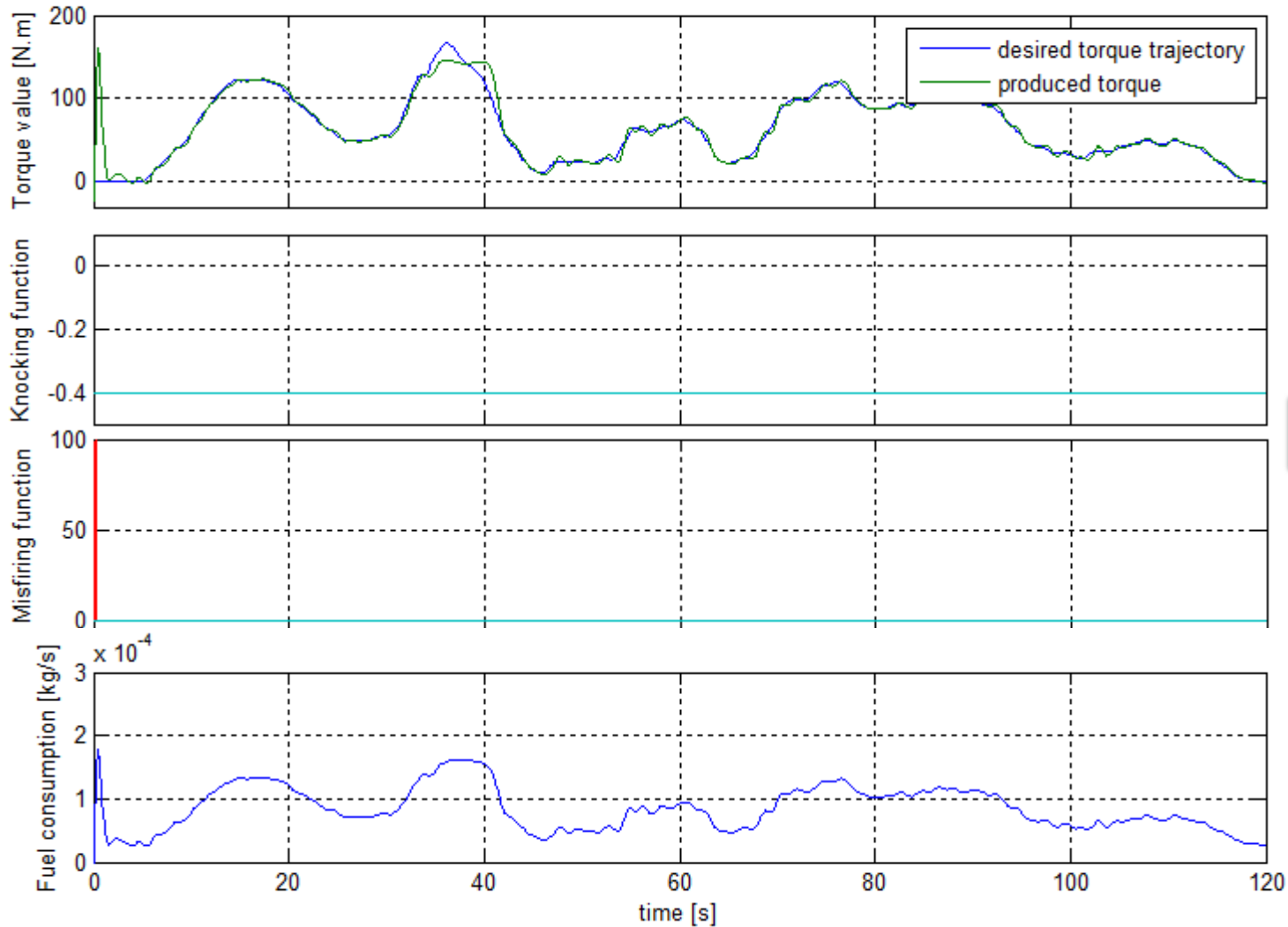
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# MPC<sub>c1,basic</sub> Scenario: output



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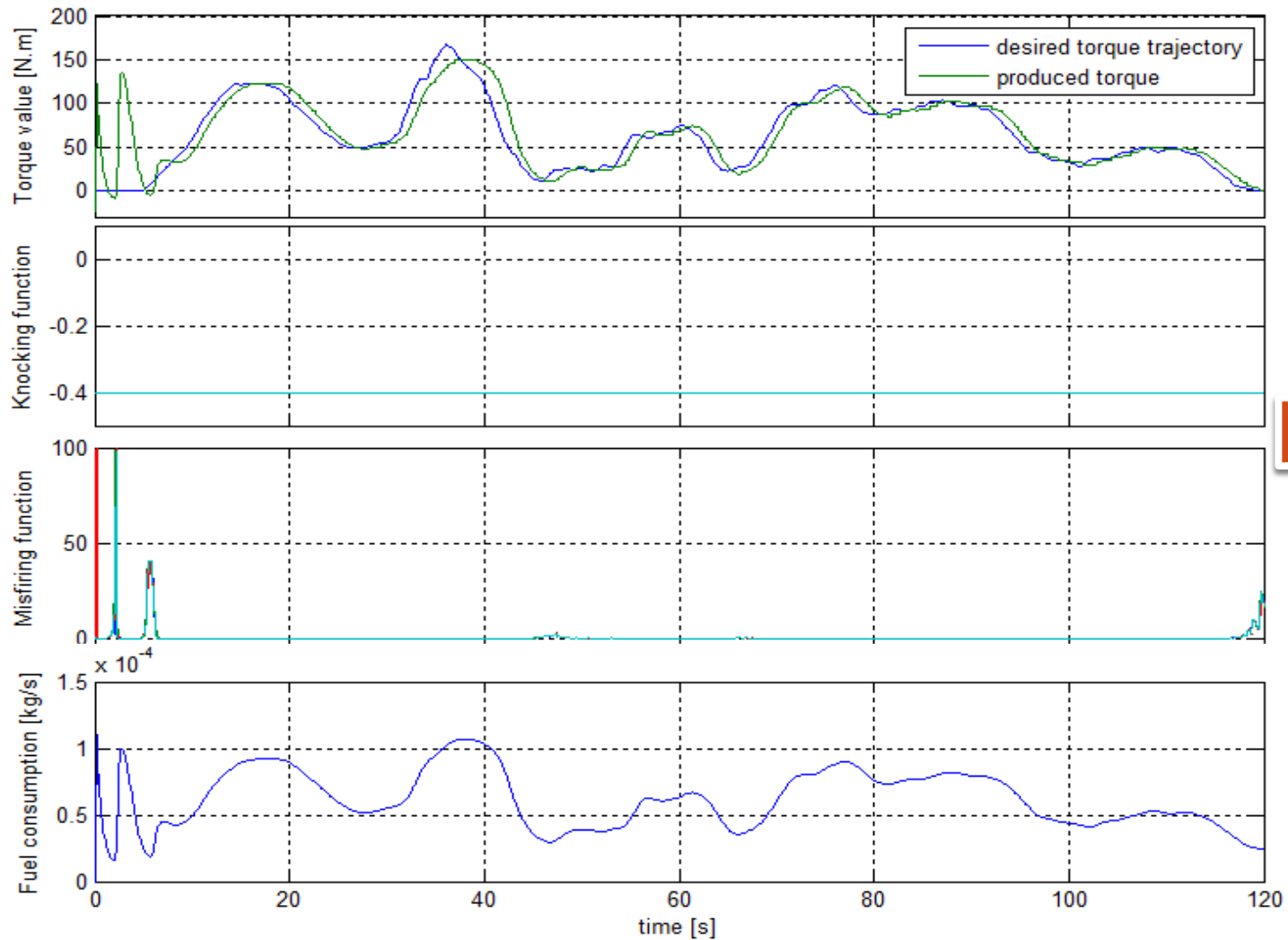
# MPC<sub>c1,advanced</sub> Scenario: output



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# MPC<sub>c2,advanced</sub> Scenario: output



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# MATLAB MPC Cost Function

$$J(z_k) = J_y(z_k) + J_u(z_k) + J_{\Delta u}(z_k) + J_\epsilon(z_k)$$

$$J_y(z_k) = \sum_{j=1}^{n_y} \sum_{i=1}^p \left\{ \frac{w_{i,j}^y}{s_j^y} [r_j(k+i|k) - y_j(k+i|k)] \right\}^2$$

$$J_u(z_k) = \sum_{j=1}^{n_u} \sum_{i=0}^p \left\{ \frac{w_{i,j}^u}{s_j^u} [u_j(k+i|k) - u_{j,target}(k+i|k)] \right\}^2$$

$$J_{\Delta u}(z_k) = \sum_{j=1}^{n_u} \sum_{i=0}^{p-1} \left\{ \frac{w_{i,j}^{\Delta u}}{s_j^u} [u_j(k+i|k) - u_j(k+i-1|k)] \right\}^2$$

$$J_\epsilon(z_k) = \rho_\epsilon \epsilon_k^2$$

$$u = \begin{bmatrix} x_{th} & 0 & 0 \\ 0 & \varphi_{SA} & 0 \\ 0 & 0 & x_{EGR} \end{bmatrix}$$

$$\Delta u = \begin{bmatrix} \Delta x_{th} & 0 & 0 \\ 0 & \Delta \varphi_{SA} & 0 \\ 0 & 0 & \Delta x_{EGR} \end{bmatrix}$$

$$r = \begin{bmatrix} T_r & 0 & 0 & 0 \\ 0 & h_{K,r} & 0 & 0 \\ 0 & 0 & h_{M,r} & 0 \\ 0 & 0 & 0 & qf_r \end{bmatrix}$$

$$y = \begin{bmatrix} T & 0 & 0 & 0 \\ 0 & h_K & 0 & 0 \\ 0 & 0 & h_M & 0 \\ 0 & 0 & 0 & qf \end{bmatrix}$$